



ME 326 - Intermediate Fluid Mechanics 

Boundary Layer

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Boundary Layer 

Outline


- Flows Past Immersed Bodies
- Boundary Layer Flows (laminar)
- Blasius Solution
- Momentum Integral Method
- Turbulent Boundary Layer Flows

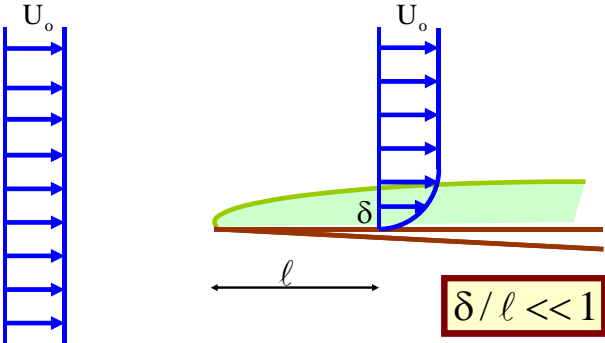
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Boundary Layer 



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Boundary Layer over a Flat Plate 



Laminar Boundary Layer

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Boundary Layer Thickness Clarkson University

Boundary Layer Thickness, δ

Distance at which \rightarrow

$\frac{u}{U_0} = 0.99$

Displacement Thickness

\rightarrow

$\delta^* = \int_0^\infty \left(1 - \frac{u}{U_0}\right) dy$

Momentum Thickness

\rightarrow

$\theta = \int_0^\infty \frac{u}{U_0} \left(1 - \frac{u}{U_0}\right) dy$

Shape Factor

\rightarrow

$H = \frac{\delta^*}{\theta}$

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Boundary Layer Theory Clarkson University

Steady Two-D Flows

$$u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right)$$

$$u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} = -\frac{1}{\rho} \frac{\partial p}{\partial y} + \nu \left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right)$$

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

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Order of Magnitude Analysis

Prandtl

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

\rightarrow

$$o\{v\} \sim \frac{\delta U_0}{l}$$

$$\frac{U_0}{l} \sim \frac{o\{v\}}{\delta}$$

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Order of Magnitude Analysis

$$u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \left(\frac{\partial^2 u}{\partial x^2} + \frac{\partial^2 u}{\partial y^2} \right)$$

$\frac{U_0^2}{l}$

\sim

$\frac{o\{p\}}{\rho l}$

\sim

$\nu \left(\frac{U_0}{l^2} + \frac{U_0}{\delta^2} \right)$

\rightarrow

$\delta \sim \sqrt{\frac{\nu l}{U_0}}$

\rightarrow

$\frac{\delta}{l} \sim \sqrt{\frac{\nu}{U_0 l}} \sim \frac{1}{\sqrt{Re_l}}$

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Order of Magnitude Analysis

$$u \frac{\partial v}{\partial x} + v \frac{\partial v}{\partial y} = -\frac{\partial p}{\partial y} + v \left(\frac{\partial^2 v}{\partial x^2} + \frac{\partial^2 v}{\partial y^2} \right)$$

$$\delta \frac{U_o^2}{l^2} \quad \frac{o\{p\}}{\rho \delta} \quad v \left(\frac{\delta U_o}{l^3} \quad \frac{U_o}{\delta l} \right)$$

$$p \sim \rho U_o^2 \quad \frac{\partial p}{\partial y} \sim \frac{\delta^2}{l^2} \rho U_o^2 \sim 0$$

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Boundary Layer Equations Clarkson University

$$u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = -\frac{1}{\rho} \frac{dp}{dx} + \nu \frac{\partial^2 u}{\partial y^2}$$

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$



Ludwig Prandtl

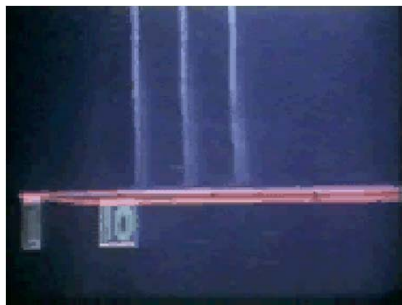
Boundary Conditions

at $y = 0$ $u = 0, v = 0$
at $y = \infty$ $u = U_o$

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Boundary Layer over a Flat Plate Clarkson University



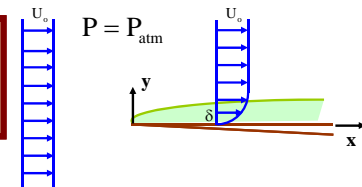
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$$u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} = \nu \frac{\partial^2 u}{\partial y^2}$$

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$



Boundary Conditions

at $y = 0$ $u = 0, v = 0$
at $y = \infty$ $u = U_o$

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Blasius Solution Clarkson University

$$\frac{u}{U_0} = f'(\eta)$$

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Blasius Similarity Solution Clarkson University

$\eta = y\sqrt{\frac{U_0}{\nu x}}$

$\frac{u}{U_0} = f'(\eta)$

$\frac{\partial u}{\partial y} = f''(\eta)\sqrt{\frac{U_0}{\nu x}}$

Blasius Equation

Boundary Layer Eq.

→

$ff'' + 2f''' = 0$

Boundary Conditions

→

at $\eta = 0$ $f = 0, f' = 0$
 at $\eta = \infty$ $f' = 1$

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Blasius Similarity Solution Clarkson University

Boundary Layer Thickness, δ

Numerical Solution

$\delta = 5\sqrt{\frac{\nu x}{U_0}}$

$\frac{\delta}{x} = 5\text{Re}_x^{-1/2}$

$\tau = \mu \frac{du}{dy}\bigg|_{y=0} = \mu U_0 f''(0) \sqrt{\frac{U_0}{\nu x}}$

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Friction Coefficient

→

$$C_F = \frac{\tau}{\frac{1}{2}\rho U_0^2} = \frac{2f''(0)}{\sqrt{\text{Re}_x}} = \frac{0.664}{\sqrt{\text{Re}_x}}$$

Drag Coefficient

→

$$C_D = \frac{D}{\frac{1}{2}\rho U_0^2 l} = \frac{4f''(0)}{\sqrt{\text{Re}_l}} = \frac{1.328}{\sqrt{\text{Re}_l}}$$

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Displacement Thickness

➔

$\delta^* = \int_0^\infty \left(1 - \frac{U}{U_0}\right) dy = 1.721 \sqrt{\frac{\nu x}{U_0}}$

Momentum Thickness

➔

$\theta = \int_0^\infty \frac{U}{U_0} \left(1 - \frac{U}{U_0}\right) dy = 0.664 \sqrt{\frac{\nu x}{U_0}}$

Shape Factor

➔

$H = \frac{\delta^*}{\theta} = 2.51$

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Momentum Integral Method

Conservation of Mass

$$\rho \int_0^h u dy + \dot{m}_{BC} - \rho \int_0^h U_0 dy = 0$$

$$\dot{m}_{BC} = \rho \int_0^h (U_0 - u) dy = \rho U_0 \delta^*$$

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Integral Method Clarkson University

Momentum Equation

$$\sum_{\text{Outlets}} \rho_o V_o A_o V_{ox} - \sum_{\text{Inlets}} \rho_i V_i A_i V_{ix} = \sum F_x$$

$$\rho \int_0^h u^2 dy + \dot{m}_{BC} U_0 - \rho \int_0^h U_0^2 dy = -D$$

$$D = \rho \int_0^h u(U_0 - u) dy = \rho U_0^2 \theta$$

von Karman Momentum Integral

➔

$$\tau_w = \rho U_0^2 \frac{d\theta}{dx}$$

von Karman

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Integral Method Clarkson University

Procedure

- Assume a velocity profile that satisfies the boundary conditions.
- Evaluate wall shear stress and θ .
- Use Momentum Integral and find δ
- Evaluate Boundary Layer parameters θ , δ , C_F , C_D .

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Concluding Remarks

- Flows Past Immersed Bodies
- Boundary Layer Flows (Laminar)
- Prandtl Boundary Layer Theory
- Blasius Solution
- Momentum Integral Method
- Turbulent Boundary Layer Flows

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Thank you!

Questions?

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